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MONTE CARLO APPROACH TO TOUCHDOWN DYNAMICS FOR SOFT LUNAR LANDING

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NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

TABLE OF CONTENTS

		Fage
SUMMARY	•	1
INTRODUC	CTION	1
VEHICLE (CHARACTERISTICS	1
INITIAL CO	ONDITIONS	4
TOUCHDO	WN DYNAMICS PROGRAM	9
RESULTS		11
APPENDIX	A. TOUCHDOWN CONDITIONS	37
APPENDIX	B. RANDOM NUMBER GENERATOR CONFIDENCE TEST	47
	LIST OF ILLUSTRATIONS	
Figure	Title	Page
1.	Vehicle Configurations	17
2.	Spring Constants for Vehicles I, II, and III	18
3.	Force-Stroke Relationship	19
4.	Chi-square Frequency Function	20
5.	Chi-square Distribution Function	21
6.	Normal Distribution Function	22
7.	Surface and Body Axes Orientation	23
8.	Symmetric Heading Orientations	24
9.	Stability Boundary for Vehicle 3	25
10.	Stability Boundary for Vehicle 7	26
11.	Lower Limit Probability of Stable Landing	27

LIST OF TABLES

Table	Title	Page
1.	Geometric Properties of LFV, LEM, and LLS	28
2.	Geometric Properties of Vehicles I, Π , and Π I	28
3.	Limiting Forces for Type I Vehicles	29
4.	Dynamic Scaling Factors	29
5.	Initial Conditions from Chi-Square Distribution	30
6.	Tabulation of Distribution Functions	31
7.	Summary of Touchdown Dynamics Runs for Vehicle 1	32
8.	Summary of Touchdown Dynamics Runs for Vehicle 5	34
9.	Summary of Cases which Tumble	35
10.	Sample Probability of Stable Landing	36
11.	Lower Limit Probability of Stable Landing	36

DEFINITION OF SYMBOLS

Symbol	Definition
C .	Confidence coefficient
CSA	Velocity cross-slope angle (angle between total horizontal velocity vector and principal slope direction)
c_{mp}	Crushing force in main strut
$^{\mathrm{C}}_{\mathrm{sp}}$	Crushing force in support strut
D	Landing gear diameter
g'	Lunar gravity
k_{V}	Vertical spring constant per leg
k_{Z}	Radius of gyration in pitch
L	Height of the center of gravity
m	Vehicle mass
$^{\mathrm{n}}\mathrm{_{f}}$	Number of cases which tumble
n	Number of cases in sample
N	Number of legs
$_{\mathrm{p}_{_{\mathbf{S}}}}^{-}$	Sample probability of stable landing
ps	Lower confidence limit on probability of stable landing for binomial population
r	Radius from vehicle's centerline to main strut attachment point
$\overline{\mathrm{v}}_{\mathrm{ht}}$	Nondimensional total horizontal velocity
\overline{v}_{v}	Nondimensional vertical velocity
v_{ht}	Total horizontal velocity

DEFINITION OF SYMBOLS (Concluded)

Symbol	Definition
v_h	Component of horizontal velocity in the direction of principal slope
v_{v}	Vertical velocity
$V_{ m z}$	Component of horizontal velocity perpendicular to direction of principal slope
$\mathbf{Y}_{\mathbf{a}}$	Distance from bottom of vehicle to main strut attachment point
$\mathbf{Y}_{\mathbf{p}}$	Distance from landing gear foot pads to bottom of vehicle
α , β , γ	Dynamic scaling factors shown in Table 4
γ	Lunar Slope
$\theta_{ m p}$	Vehicle's pitch angle referenced to the horizontal
o_0	Initial pitch angle with respect to lunar surface
ϕ 0	Vehicle's initial heading orientation with respect to principal slope direction
ϕ 03	Equivalent heading orientation for three-legged vehicle
ϕ 04	Equivalent heading orientation for four-legged vehicle
σ	Standard deviation for normal distribution
u	Degrees of freedom for chi-square distribution
ω	Angular velocity
ψ_0	Initial bank angle

MONTE CARLO APPROACH TO TOUCHDOWN DYNAMICS FOR SOFT LUNAR LANDING

SUMMARY

Results of analytical touchdown dynamics investigations are presented which were conducted to obtain estimates of the probability of stable landing for configurations with various landing gear diameters. Both three-legged and four-legged vehicles are considered in the analysis. Dynamic scaling considerations are taken into account so that the results are applicable to a wide range of vehicle size and mass. A Monte Carlo approach is taken in the determination of initial landing conditions.

Results indicate that for a given probability of stable landing, a three-legged vehicle requires a landing gear diameter only slightly larger than the diameter required for a four-legged vehicle. Therefore the three-legged vehicle's landing gear should weigh less.

INTRODUCTION

In the overall design and performance of spacecraft intended to soft-land on the moon, analysis of touchdown dynamics motion is an important part. Such analysis in the past has been restricted to specific missions and vehicles designed for those missions such as Surveyor (ref. 1), Lunar Excursion Module (ref. 2), Lunar Flying Vehicle (ref. 3), and Lunar Logistics System (ref. 4). In general, the analysis for each vehicle has not been a complete parametric study over a wide range of initial conditions, but rather a limited study with emphasis on what the vehicle can do under the worst set of conditions. The parametric approach is not feasible because of the large number of parameters involved. For example, if a parametric study of a particular vehicle is considered with only three variations of ten initial touchdown conditions, a total of three to the tenth power, or 59,049 combinations, is produced.

Presented in this report are results of a general touchdown dynamics study in which a Monte Carlo approach is taken to the determination of initial conditions. This approach is more realistic than simply choosing the worst-case conditions and does not require the prohibitive amount of analysis which a

parametric approach would require. The vehicle configurations used are determined in a standardized form, and the results are equally applicable to the Lunar Flying Vehicle, Lunar Excursion Module, or Lunar Logistics System, even though these vehicles vary greatly in size and mass.

Initial conditions have been determined using a Monte Carlo approach for 400 cases. For each case, the initial conditions were determined for 10 parameters: vertical velocity, horizontal velocity, velocity cross-slope angle, vehicle pitch, bank, and heading angles, lunar slope, and the vehicle's pitch, roll, and yaw rates at touchdown. The coefficient of friction between the landing pads and lunar surface is assumed to be infinite (no sliding). Eight generalized vehicles are analyzed for these 400 cases to determine whether or not the vehicle being considered lands safely or tumbles. The eight vehicles considered correspond to three-legged and four-legged vehicles with four variations in the ratio of landing gear diameter to center-of-gravity height. From these results, an estimate is made of the probability of stable landing on the lunar maria for each of the eight vehicles.

The author is indebted to Mr. John D. Capps, Computation Laboratory, who programmed the equations for the three-dimensional touchdown dynamics digital program as well as the program used in the determination of initial conditions.

VEHICLE CHARACTERISTICS

The Lunar Flying Vehicle (LFV) is a small vehicle which has received consideration for lunar surface exploration after the initial Apollo landings. Primary emphasis has been placed upon a vehicle having an 80-kilometer maximum range when carrying a payload of two men with their portable life support systems. The primary mission for such a vehicle was considered to be rescue back to the Lunar Excursion Module (LEM) from a disabled surface roving vehicle. A secondary mission was that of supplementing the roving vehicle by permitting flights into areas which are inaccessible to a roving vehicle.

The Lunar Logistics System (LLS) is a large cargo-carrying vehicle which has received study as a means of delivering 15-ton payloads to the moon in support of a lunar base.

Even though the three spacecraft (LFV, LEM, LLS) vary greatly in size and mass, all three vehicles can be reduced to approximately the same nondimensional geometry as shown in Table 1. As a result of this, three standard vehicles (I, II, and III) have been established which are representative of the LFV, LEM,

and LLS size vehicles. The standardized dimensions are shown in Table 2. The radius of gyration about the bank (roll) axis is assumed equal to that shown for pitch. The radius of gyration in yaw (about the vehicle's centerline) is taken to be 0.9 times the pitch value. Having standardized the three vehicles, all touchdown dynamics calculations performed for Vehicle I can be made applicable through dynamic scaling factors to Vehicle II and Vehicle III.

Eight vehicles are considered, as shown in Figure 1, corresponding to four values of the ratio of landing gear diameter to center-of-gravity height. Both three-legged and four-legged vehicles are considered even though the LFV, LEM, and LLS have all received major emphasis as four-legged vehicles. A major purpose of this study, however, is to obtain estimates of the relative probability of stable landing with three-legged vehicles compared to four-legged vehicles. Although the diameter--c.g. height ratios for LFV, LEM, and LLS vary from 2.50 to 3.25 (Table 1), the values chosen for this generalized study vary from 2.0 to 3.0. The large landing gear diameter designed for the LLS was based upon very severe initial conditions which are believed to be overly conservative, resulting in an excessively heavy landing gear.

The spring rates assumed for the vehicles of this study (Fig. 2) are the equivalent vertical spring rates per leg when landing on a level, high-friction surface with no sliding of the foot pads. These vertical spring rates were obtained by assuming an elastic spring rate (7005 N/cm or 4000 lb/in for Vehicle I) along the three struts of an inverted tripodal leg. As seen, the vertical spring rate decreases as the landing gear diameter increases, negating somewhat the beneficial result of the larger landing gear diameters. The spring rates assumed for the LLS size vehicle (Type III) are rather large and may be very difficult to obtain. These spring rates are required to satisfy dynamic similarity. The results of this study may therefore be somewhat optimistic for this larger vehicle because of the high stiffness assumed. On the other hand, the assumed spring rate for Vehicle I (LFV) is probably low so that the results obtained may be somewhat pessimistic. Previous study (ref. 5) has shown that decreasing stiffness has a detrimental effect on landing stability.

The force-stroke relationship in each strut is shown in Figure 3. Upon compression, each strut is assumed to compress elastically until the limiting force in compression is reached. Further stroking takes place with a constant stroking force assumed. As the strut begins to re-extend, the stored elastic energy is released until the strut force reaches zero. Further re-extension of the strut takes place under zero load or free-return conditions. No limiting force in tension is assumed. The same spring constant is assumed for the strut in tension as used in compression. The limiting force is assumed to represent the crushing force of aluminum honeycomb energy absorbers in the struts. Upon subsequent stroking, the strut shortens under zero load as long as it is still in the free-return stroking region.

The limiting force or crushing force values for the eight vehicles were determined so that the deceleration load factor is limited to 3.0 earth g's when the vehicle lands on a level, high friction surface on all legs simultaneously. This load factor appears to be a good choice from previous study (ref. 6). If the chosen load factor is too high, a landing gear designed for such loads is too heavy. If the load factor is too low, excessive stroking is required to absorb the landing energy, thus increasing the strut length required and landing gear weight. Each leg is considered to be an inverted tripodal arrangement of one main strut and two lower support struts. The limiting force of each support strut is considered to be one—third of the main strut limiting force. This relationship is believed to be a good choice based upon previous analysis (ref. 7). The limiting forces obtained for the eight vehicles are shown in Table 3. These values correspond to the forces for Vehicle I (LFV). Corresponding forces for the larger vehicles are increased by the dynamic similarity force factors given in Table 4.

The mass of Vehicle I (LFV) is considered to be $453.59 \, \mathrm{kg}$ (earth weight of $4448 \, \mathrm{N}$ or $1000 \, \mathrm{lb}$). The mass ratios given in Table 4 can be used to obtain the mass of the larger vehicles.

INITIAL CONDITIONS

Initial conditions have been obtained using a Monte Carlo approach which randomly selects values from zero to one and relates these values to the initial conditions sought through the distribution functions that have been established for the initial variables. Determining a reasonable distribution function for each of the variables of interest is the most questionable part of the entire study because of the lack of data. However, one value of this study may be that of helping to point out the need for detailed and realistic statistical information concerning initial touchdown conditions.

Various simulations have been performed (refs. 8-13) which have yielded some information on the conditions at touchdown for lunar landing. All of these simulations have been conducted with varying degrees of realism relative to instrumentation and control response, pilot training, test objectives, pilot motivation, and other factors. Results must be viewed with some degree of caution since they may not realistically simulate an actual vehicle with well trained and motivated pilots. Future efforts, it is hoped, will provide more realistic information on touchdown conditions through the use of the Lunar Landing Research Facility (ref. 14) and also the free flight Lunar Landing Vehicle (ref. 15).

As far as the vertical and horizontal components of touchdown velocity are concerned, it is reasonable to expect that the vast majority of cases of actual landings with well motivated pilots will be very soft landings with low touchdown

velocities. The simulations reported by Hill (ref. 9) show that the vertical descent rate is less than 1.2 m/s (4 fps) for 90 percent of the cases. The forward velocity at touchdown was also reported as less than 0.6 m/s (2 fps) for 90 percent of the cases. The simulation results reported by Wood and Post (ref. 12) indicate more severe landing velocities. About 35 percent of the time the pilots failed to land within the design envelope of 3.0 m/s (10 fps) vertical velocity, and 1.5 m/s (5 fps) horizontal velocity. It was concluded that this large failure rate can be attributed not only to difficulty of the task, but also to the training level of the pilots and limitations in the simulation. It was believed that greater pilot experience, and improvements in displays and display precision will reduce such failures considerably in future studies.

It is expected that the frequency functions of the vertical and horizontal velocity components should have some tail-off toward higher velocities to represent more severe cases. The frequency function chosen as a reasonable representation of the vertical and horizontal velocity components is given in Figure 4. This function corresponds to the chi-square frequency function with three degrees of freedom. Appropriate choices for multiplication constants convert the values of the variable x to the vertical and horizontal velocities.

Concerning the lunar slope expected in maria regions of the moon, it is expected that the vast majority of landings will be on low sloped terrain with only a small percent of the landings on rougher terrain. It may well be argued that manned landings on rough, highly sloped terrain will never take place. However, for the small LFV exploration system, some landings may well be on steeper slopes because of the desire to land near interesting geological and topological features. In addition, if lunar dust somewhat clouds the pilot's vision during the final phase of landing, it is possible that one or more pads may impact on a protuberance or into a depression which would increase the effective landing slope. In addition, the pad-surface interaction during impact may well serve to increase the effective landing slope. Experimental evidence (refs. 16-17) indicates that the bearing strength of the soil is not the sole criterion for penetration. There is a tendency for the downhill pads to penetrate the soil more deeply than the uphill pads during a landing, and thus increase the effective landing slope. Considering these factors, it is assumed that the frequency function shown in Figure 4 is also a reasonable approximation of the effective slope to be encountered upon landing. Mason and colleagues (ref. 18) have established a similar highly skewed frequency function for a representative lunar surface profile from 20 100-kilometer traverses. Their work was performed to establish a baseline for determining lunar roving vehicle requirements. An appropriate choice of a multiplication factor converts the variable x in Figure 4 into the effective lunar slope.

The distribution (cumulative frequency) function, shown in Figure 5, is used in the Monte Carlo approach to obtain vertical and horizontal velocity components and the effective lunar slope. The values given in Table 5 have been obtained

from this distribution function by use of appropriate multiplication constants. The multiplication constants were chosen on the basis of the values emphasized in Table 5. For example, McCauley indicates (ref. 19) that, in the 1-10 meter scale, the median slope of the maria is about 5 degrees. Thus, with the assumed distribution, 20 percent of the maria has a slope of about 2 degrees or less, 80 percent is within 10 degrees, and 1 percent is 24 degrees or more.

For the LEM size (Type II) vehicle, it is assumed that 99.865 percent of the landings will have a vertical velocity of 3.048 m/s (10 fps) or less, and a horizontal velocity of 1.524 m/s (5 fps) or less. Thus, with the assumed distribution, 90 percent of the landings have a vertical velocity of 1.188 m/s (3.90 fps) or less, and a horizontal velocity of 0.594 m/s (1.95 fps) or less. While the values were based upon the LEM design values of 10 fps and 5 fps (ref. 12), the 90 percentile values also correspond quite well with the simulation results of Hill (ref. 9). Thus, it is considered that the LEM size vehicle (Type II) will land 50 percent of the time with a vertical velocity of 0.448 m/s (1.47 fps) or less, with a horizontal velocity of 0.224 m/s (0.735 fps) or less.

For the results to be equally applicable to the LFV size (Type I) vehicle and the LLS size (Type III) vehicle, the velocities (also shown in Table 5) were adjusted by dynamic scaling considerations. For example, it is assumed that 90 percent of the landings for the LLS (Type III) vehicle will be with a vertical velocity of 1.638 m/s (5.374 fps) or less. For the LFV (Type I) vehicle, it is assumed that 90 percent of the landings will occur with a vertical velocity of 0.706 m/s (2.32 fps) or less.

The dynamic scaling factors given in Table 4 show that, for dynamic similarity, the velocity is scaled as the square root of the vehicle's linear dimension when landing in the same gravity field. Thus, for larger vehicles, larger landing velocities can be tolerated. While this indicates that the larger vehicles may be easier to land, this table also shows that the spring rate for dynamic similarity is proportional to the mass and inversely proportional to the linear dimension when landing in the same gravity field. Thus, since the mass increases to some power of the linear dimension, higher spring rates are required for the larger vehicles for dynamic similarity.

The horizontal velocity is assumed to be controlled as far as possible to be zero. Therefore, it is considered that the direction of the horizontal velocity vector at touchdown with respect to the principal slope direction has an equal probability of being in any direction. Further study of this problem may show an advantage to having a horizontal velocity of sufficient magnitude to guarantee an uphill landing rather than a smaller residual component that may be in the downhill direction. The assumption of trying to land at zero horizontal velocity, however, should certainly be applicable to unmanned vehicles such as the LLS or an unmanned LEM used as a LEM truck. It is also assumed that the vehicle's

geometric heading orientation with respect to the principal slope direction has an equal probability of being in any direction. This assumption is applicable to unmanned vehicles and also to manned vehicles where vision impairment due to lunar dust might make the determination of the direction of principal slope difficult. Under clear vision, the pilot would probably orientate the vehicle's heading to align in a 1-2-1 orientation to the slope for improved stability.

The pitch and bank angles at touchdown, referenced to the horizontal, are assumed to follow a normal distribution with zero mean. Likewise, the vehicle's angular rates in pitch, roll, and yaw are also assumed to follow the same distribution. The normal distribution with zero mean and standard deviation of one is shown in Figure 6. Appropriate multiplication constants convert the x variable to angles and angular rates. The standard deviation for pitch and bank angles is considered to be 2 degrees and is applicable for Vehicles I, II, and III. The standard deviation of pitch, roll, and yaw rates is taken to be 1.60 degrees per second for the Type II (LEM size) vehicle. Past simulation studies show that these are reasonable assumptions. For dynamic similarity, the corresponding standard deviation of angular rate is taken to be 2.70 degrees per second for the Type II (LFV) vehicle and 1.16 degrees per second for the Type III (LLS) vehicle.

The 10 variables of vertical velocity, horizontal velocity, lunar slope, cross-slope angle, heading angle, bank angle, pitch angle, pitch rate, roll rate, and yaw rate are assumed to be uncorrelated.

The Monte Carlo procedure for obtaining the initial conditions can be summarized in the following manner. A set of values for 400 cases is obtained where each case consists of obtaining 10 pseudo-random numbers from a previously developed digital random number generator subroutine for a rectangular frequency function (ref. 20). These numbers are equal to or greater than zero and equal to or less than one. These 10 random numbers, RN_1 through RN_{10} , are then used as $F(x_1)$ through $F(x_{10})$ to obtain the values for x_1 through x_{10} from Table 6.

This table was obtained from values tabulated by Abramowitz and Stegun (ref. 21). Linear interpolation is used to obtain the values of x_1 to x_{10} . The initial conditions are then computed:

$$\overline{V}_{ht} = 0.0605 x_1$$
 (1)

$$V_{ht} = \sqrt{g k_z} \overline{V}_{ht}$$
 (2)

$$\overline{V}_{V} = 0.121 x_{2}$$
 (3)

$$V_{V} = \sqrt{g \, k_{Z}} \, \overline{V}_{V} \tag{4}$$

$$\gamma = 2.113 x_3$$
 (5)

$$CSA = 360 x_4$$
 (6)

$$V_{h} = V_{ht} \cos CSA \tag{7}$$

$$V_{z} = -V_{ht} \sin CSA \tag{8}$$

$$\phi_0 = 360 x_5 \tag{9}$$

$$\psi_0 = 2 x_6 \tag{10}$$

$$\theta_{\rm p} = 2 x_7 \tag{11}$$

$$\theta_0 = \theta_D + \gamma \tag{12}$$

$$\omega_{\mathbf{y}} = 2.7 \, \mathbf{x}_8 \tag{13}$$

$$\omega_{V} = 2.7 x_{9} \tag{14}$$

$$\omega_{Z} = 2.7 \, \mathrm{x}_{10}$$
 (15)

The values were obtained corresponding to the Type I (LFV) vehicle for convenience in performing touchdown dynamics computations since the time required to obtain the touchdown motion is less. A computing interval of 0.002 sec was used in performing the touchdown dynamics computations. The same results could have been obtained with the larger vehicles in the same computation time by increasing the computing interval by the appropriate time factors given in Table 4. Tabulation of the initial conditions for the 400 cases is given in Appendix A.

A random number confidence test was performed to judge the adequacy of the pseudo-random numbers generated by the digital subroutine. The results are given in Appendix B. Four thousand random numbers were generated corresponding to 400 cases with 10 random numbers in each case, RN_1 through RN_{10} , used to obtain the 10 initial conditions. Therefore, 400 values were obtained for each of the 10 random numbers RN_1 through RN_{10} . The range from zero to one was divided into 10 intervals so that these 400 values would have an expected frequency of 40 in each interval. The observed frequencies were obtained and the chi-square value obtained. These chi-square values should follow a chi-square distribution with 9 degrees of freedom. Therefore, the value of chi-square could be expected to exceed 12.2 (ref. 21) for 20 percent of the cases, or for 2 of the 10 random numbers. None of the random numbers have a chi-square value which exceeds the 12.2 value. Therefore, the values generated by the random number subroutine are adequate.

TOUCHDOWN DYNAMICS PROGRAM

The touchdown dynamics program used for this study corresponds to a three-dimensional method programmed for digital computer operations. The method was developed as a simplified version of the landing dynamics method of analysis for the LEM described by Mantus, Lerner, and Elkins (ref. 22). This simplified version is restricted to landings on a plane sloping surface without protuberances or depressions and without sliding of the landing pads along the surface. The surface and body axes orientation used is shown in Figure 7. The velocity cross-slope angle is the angle from the principal slope direction to the total horizontal velocity vector. The vehicle's heading angle defines the orientation of the vehicle's landing pads. Zero heading angle corresponds to a 1-2-1 pad orientation for a four-legged vehicle and to a 1-2 pad orientation for a three-legged vehicle. Heading angles of ±45 degrees correspond to a 2-2 pad orientation for the four-legged vehicle. Heading angles of ±60 degrees correspond to a 2-1 pad orientation for the three-legged vehicle. These various orientations are shown in Figure 8.

As previously stated, the initial heading orientation has been taken to have equal probability of being in any direction (360 degrees). However, as far as the touchdown dynamics program is concerned, 0, 90, 180, 270, and 360 degrees heading orientation is still a 1-2-1 orientation for a four-legged vehicle. Similarly, 0, 120, 240, and 360 degrees heading orientation is still a 1-2 orientation for a three-legged vehicle. Consequently, the initial heading angle obtained from the Monte Carlo analysis has been converted to an equivalent heading within ± 45 degrees for the four-legged vehicle and within ± 60 degrees for the three-legged vehicle for input into the touchdown dynamics program. Thus:

$$\phi_{04} = \phi_0$$
 , $0 \le \phi_0 \le 45$ (16)

$$\phi_{04} = \phi_0 - 90 , \qquad 45 < \phi_0 \le 135 \tag{17}$$

$$\phi_{04} = \phi_0 - 180, \qquad 135 < \phi_0 \le 225$$
 (18)

$$\phi_{04} = \phi_0 - 270, \qquad 225 < \phi_0 \le 315 \tag{19}$$

$$\phi_{04} = \phi_0 - 360, \qquad 315 < \phi_0 \le 360$$
 (20)

and

$$\phi_{03} = \phi_0$$
 , $0 \le \phi_0 \le 60$ (21)

$$\phi_{03} = \phi_0 - 120, \qquad 60 < \phi_0 \le 180$$
 (22)

$$\phi_{03} = \phi_0 - 240, \qquad 180 < \phi_0 \le 300$$
 (23)

$$\phi_{03} = \phi_0 - 360, \quad 300 < \phi_0 \le 360$$
 (24)

These equivalent heading angles for the 400 cases are also tabulated in Appendix A.

Each run begins with the vehicle's lowest pad just touching the surface. Strokes and forces along the struts are computed with subsequent determination of forces and moments referred to the body axes. The moments are used to obtain the angular accelerations about the body axes and are integrated numerically to obtain the angular velocities about the body axes. The rate of change of the angles θ , ψ , and ϕ are then computed from expressions containing the angular velocities about the body axes. The angular accelerations of $\ddot{\theta}$, $\ddot{\psi}$, $\ddot{\phi}$ are then obtained by equations derived from differentiating the equations for $\dot{\theta}$, $\dot{\psi}$, $\dot{\phi}$. The angles θ , ψ , and ϕ are obtained by numerical integration using the computed rates and angular accelerations for these angles. The forces computed are converted to forces along the inertial (surface) axes and the vehicle's translational accelerations obtained. These accelerations are integrated numerically to obtain the velocity and position of the vehicle's center of gravity. The computations are repeated at each time step until sufficient time has elapsed to determine whether or not the vehicle tumbles or is stable.

As long as a pad is off the surface, it travels along with the vehicle. Upon reaching the surface, the pad remains at its impact position until the forces on the pad normal to the surface become negative. The negative force is set to zero and the pad then lifts from the surface and again travels along with the vehicle until it impacts the surface again.

Examples of stability boundaries which have been obtained by this program are shown in Figures 9 and 10. Figure 9 gives the stability boundary in terms of the lunar slope versus the vehicle's heading angle for a three-legged vehicle. Figure 10 gives corresponding results for a four-legged vehicle. As seen from Figure 9, stable landings can be made on lunar slopes from 6 degrees to 18 degrees depending upon the vehicle's heading orientation with respect to the principal slope direction. For the four-legged vehicle (Fig. 10), stable landings can be made on slopes from 10 degrees to about 18 degrees. Minimum stability does not correspond to 45 degrees heading (2-2 orientation) but rather to a heading angle of about 30 degrees for this case. The impact velocity values shown correspond to the small (Type I) vehicle.

RESULTS

With the three-legged vehicle having the smallest landing gear (Vehicle 1), the 400 cases were reviewed to determine by inspection which cases were more likely to tumble. Most of the cases are obviously stable because of the small touchdown velocities, combined with small surface slopes. Three of the cases were considered to obviously tumble because of the surface slope being greater than 24 degrees. Touchdown dynamics motion was then computed for 45 cases which were believed to have a substantial probability of tumbling. Of these, 14 cases did result in tumbling motion.

Based upon these results, touchdown motion was computed for an additional 34 cases which were believed to be stable, but with a substantial degree of uncertainty. Of these, three cases tumbled. Finally, an additional nine cases were chosen which were thought definitely would be stable but lacked the high degree of certainty desired. Of these, none tumbled. Based upon these results, a final review was made of the remaining cases with the conclusion that all would certainly be stable. Therefore, for Vehicle 1, 20 cases of the 400 correspond to tumbling cases.

A summary of the cases used for the touchdown dynamics computations for Vehicle 1 is given in Table 7. It is observed that tumbling resulted for Case 43 with a lunar slope of only 6.9 degrees while Case 332, with a lunar slope of 19.2 degrees, was stable. While the lunar slope is a very important parameter to consider in touchdown dynamics analysis, other initial conditions are also obviously important. Case 43 corresponds to a near 1-2 landing orientation with a relatively high vertical impact velocity, while Case 332 corresponds to a near 2-1 orientation with a small impact velocity.

Touchdown dynamics motion was then computed using Vehicle 2 for the 20 cases which were previously determined as tumbling cases. All stable cases for Vehicle 1 would also be stable cases for Vehicle 2 since nothing is changed except the landing gear diameter, which is larger. Of these 20 cases, 11 cases tumbled including the 3 cases considered obvious for Vehicle 1.

Touchdown motion was then computed using Vehicle 3 for the 11 cases which tumbled when using Vehicle 2. Of these, four cases tumbled.

Touchdown motion was then computed using Vehicle 4 for the four cases which tumbled when using Vehicle 3. Of these, none tumbled.

A similar analysis was performed for the four-legged vehicles. Vehicle 5 was used and touchdown dynamics was computed for 25 cases which were thought to have a substantial probability of tumbling. Of these, 11 cases tumbled.

Computations were then performed for 32 additional cases which were believed to be stable, but with a substantial degree of uncertainty. Of these, three cases tumbled. Finally, an additional 14 cases were considered which were thought definitely would be stable but lacked the high degree of certainty desired. Of these, none tumbled. Three cases (with a slope of over 24 degrees) were considered to be obviously tumbling cases. Therefore, for Vehicle 5, 17 of the 400 cases correspond to tumbling cases.

A summary of the cases used for touchdown dynamics calculations for Vehicle 5 is given in Table 8. No case tumbled corresponding to a lunar slope of less than 15.5 degrees, while no case was stable with a slope greater than 19.3 degrees. This is a slope spread of only 3.8 degrees compared to a slope spread of 12.3 degrees for the corresponding three-legged vehicle (Vehicle 1). While 7 cases tumbled for Vehicle 1 with a lunar slope of less than 15.5 degrees, only 13 cases tumbled with lunar slopes of 15.5 degrees or more compared to the 17 cases for Vehicle 5. These results indicate that a relatively well defined critical slope can be established for four-legged vehicles compared to three-legged vehicles.

The vehicle with the next largest diameter (Vehicle 6) was then considered. Touchdown motion was computed for the 17 cases with the result that 7 cases tumbled including the 3 cases considered obvious for Vehicle 5. These seven cases were then considered using Vehicle 7 with the result that three cases tumbled. Finally, these three cases were considered using Vehicle 8 with the result that none tumbled. The cases corresponding to tumbling motion are summarized in Table 9.

A summary of the sample probabilities for stable landing is given in Table 10. The sample probability of stable landing varies from 0.950 for the three-legged vehicle with the smallest landing gear to 1.000 for the vehicles with the largest landing gear. The lower limit probability for stable landing with 0.95 and 0.995 confidence coefficients for the binomial population has been taken from published tables (ref. 23) based upon the sample size of 400. These results are given in Table 11. The table also shows the lower limit probability for stable landing based upon an approximate equation given by Dalton (ref. 24). This equation is

$$p_{s} = \left[\frac{(3+2C)/4}{(n-n_{f}/2)}, \frac{(1-C)/e}{(1-C)/e} \right]$$
(25)

where

$$\delta = 0$$
 for $n_f = 0$

$$\delta = 1$$
 for $n_f \neq 0$.

This equation is seen to give a very good approximation of the results for the binomial population.

The lower limit probability for stable landing for the binomial population with a confidence coefficient of 0.995 is shown in Figure 11. Since neither Vehicle 4 nor Vehicle 8 had any cases which tumbled, touchdown dynamics runs were also made for an additional three-legged vehicle and an additional four-legged vehicle with a diameter-c. g. height ratio of 2.75. The three-legged vehicle tumbled for Case 317. This case has the highest lunar slope (28.2 degrees) of the 400-case sample and the vehicle lands in a near 1-2 orientation. The resulting probability for stable landing for this vehicle is also shown in Figure 11. The four-legged vehicle tumbled for Case 129 and also for Case 317. Case 129 has the second highest lunar slope (27.3 degrees) of the 400-case sample. The resulting probability for stable landing is also shown in Figure 11.

The probability values for the three-legged vehicles vary in a smooth manner, whereas the values for the four-legged vehicles are somewhat erratic. At a diameter-c. g. height ratio of 2.75, the probability of stable landing for the four-legged vehicle is less than for the three-legged vehicle based upon the 400-case sample. It is believed that this result would not be obtained if the sample size were increased by an order of magnitude. Similarly, at a diameter-c. g. height ratio of 2.25, the probability of stable landing for the four-legged vehicle appears to be somewhat high. Consequently, the curve shown in Figure 11 for the four-legged vehicles has been faired in a manner to remove the erratic nature of the data points.

The results indicate that, for a given probability of stable landing, a three-legged vehicle requires a landing gear diameter only slightly larger than the diameter required for a four-legged vehicle. Since each leg is folded for flight to the moon and must be deployed before landing, the reliability of landing gear deployment should be greater for a three-legged vehicle. With only three landing pads to contact the surface, there is less chance of contacting a protuberance or depression. A three-legged vehicle assures a more positive final resting support, whereas a four-legged vehicle may tend to rock back and forth. With only a slightly larger diameter required, the three-legged vehicle's landing gear should weigh less.

George C. Marshall Space Flight Center, National Aeronautics and Space Administration, Huntsville, Alabama, September 30, 1965

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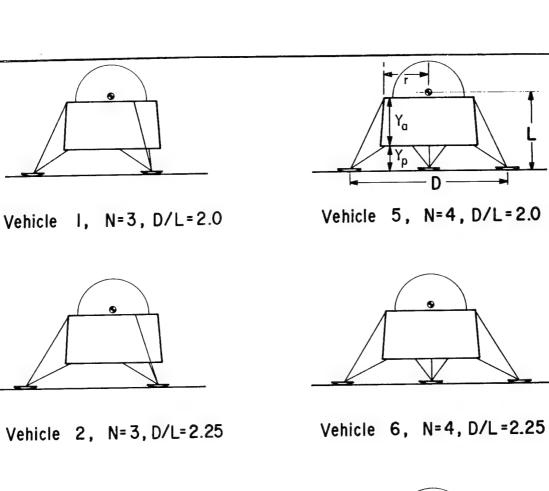
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Vehicle 3, N=3, D/L=2.5 Vehicle 7, N=4, D/L=2.5

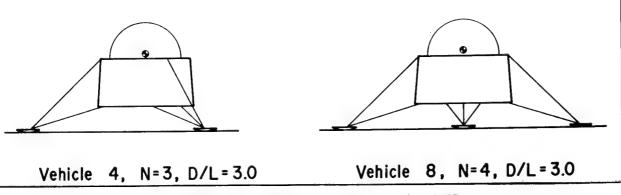


FIGURE 1. VEHICLE CONFIGURATIONS

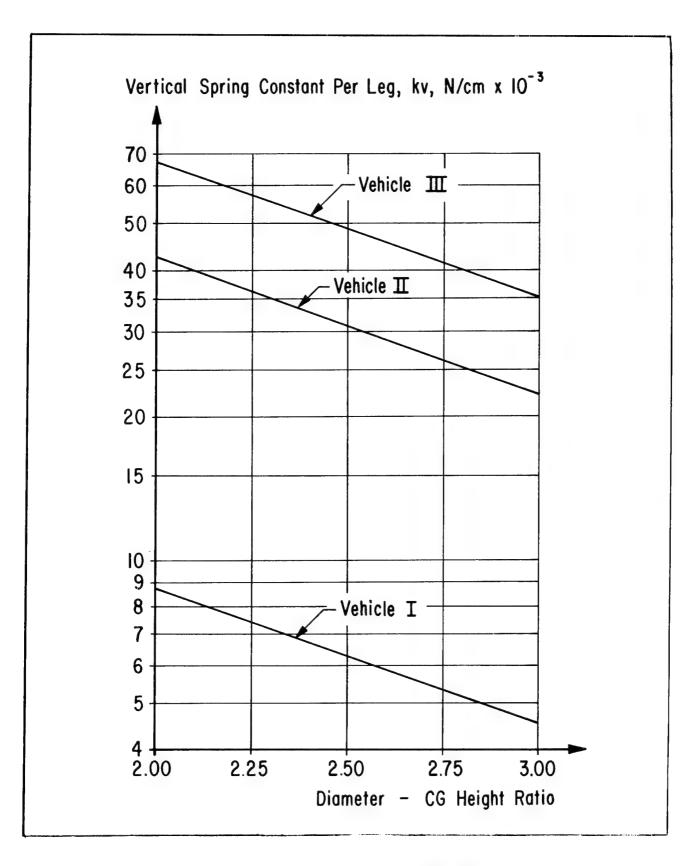


FIGURE 2. SPRING CONSTANTS FOR VEHICLES I, II, AND III

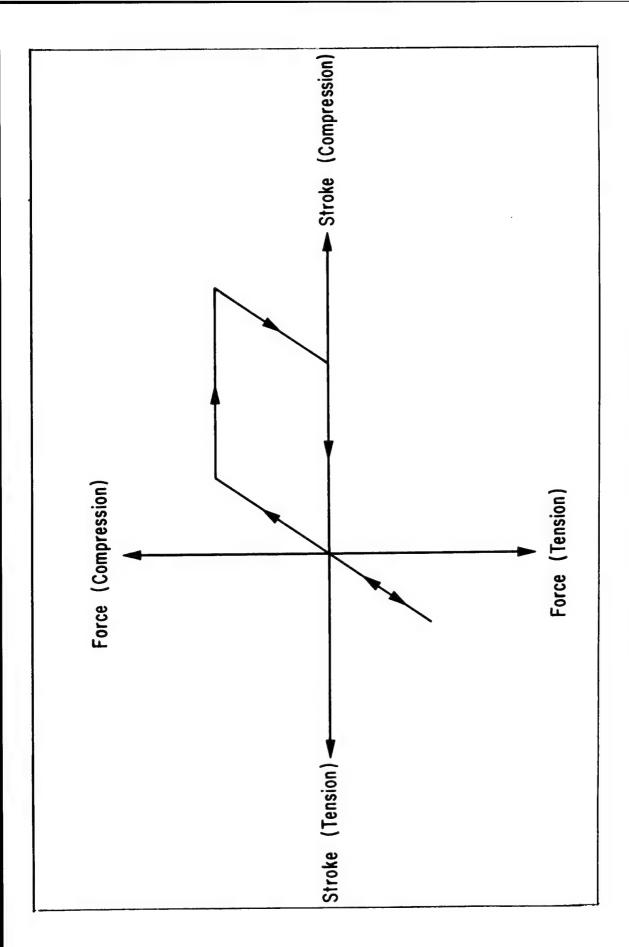


FIGURE 3. FORCE-STROKE RELATIONSHIP

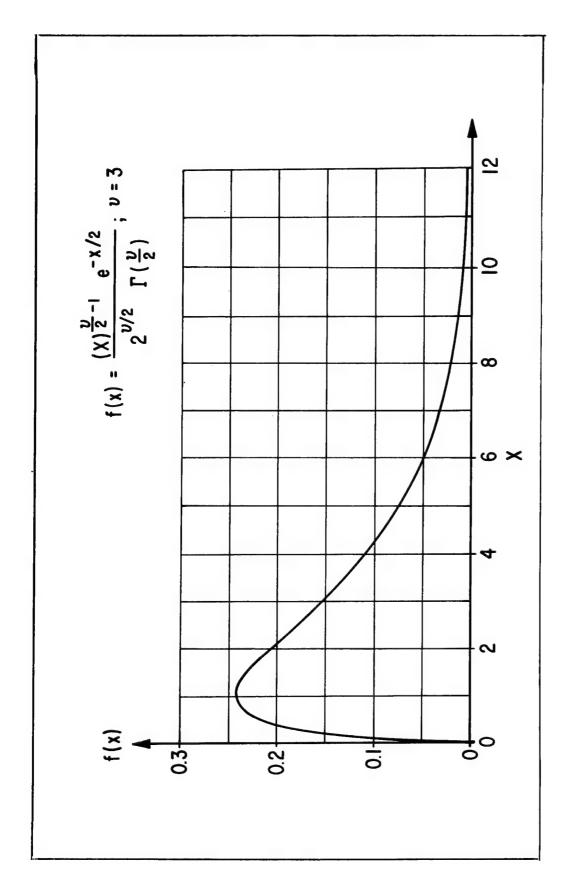
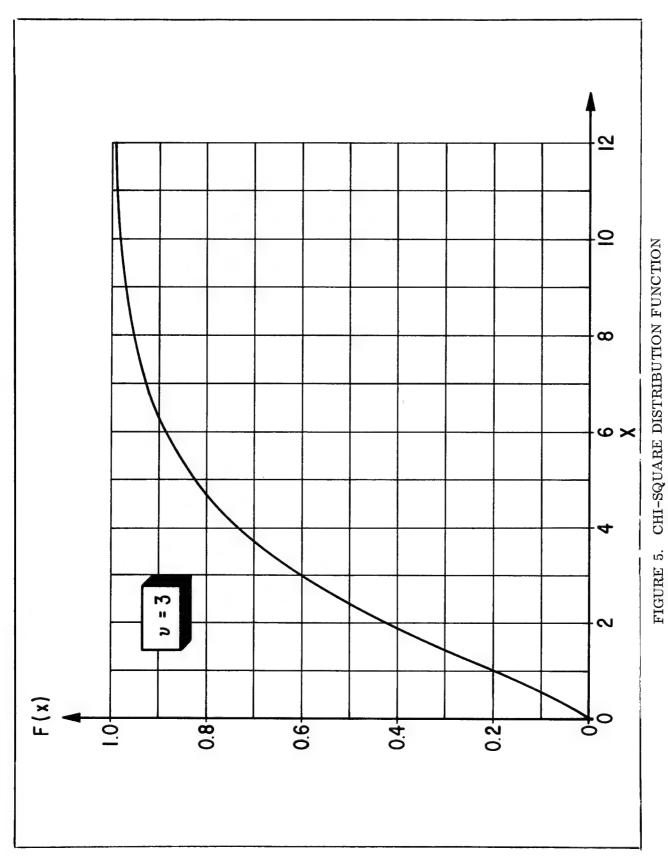
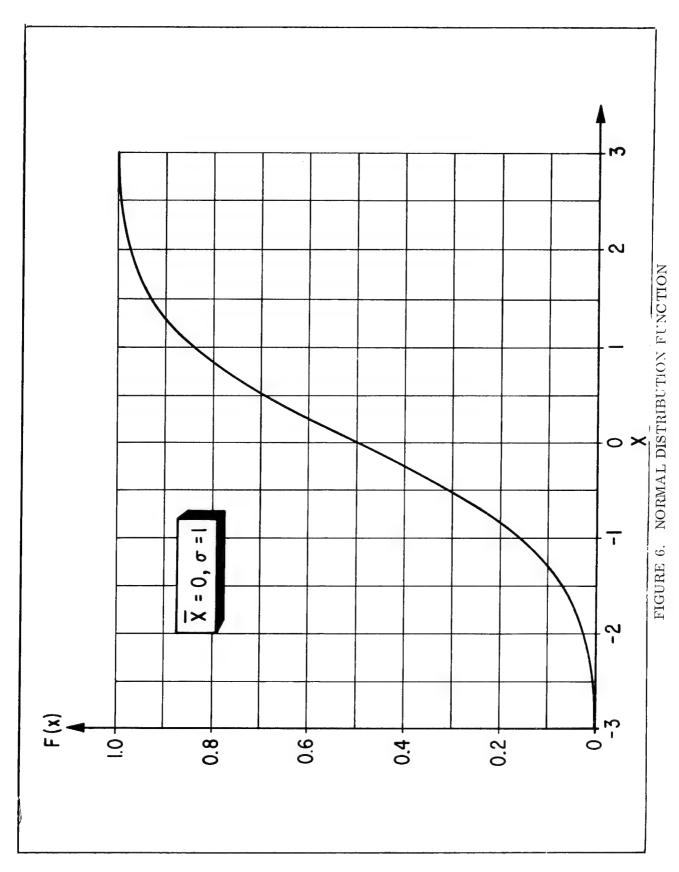


FIGURE 4. CHI-SQUARE FREQUENCY FUNCTION





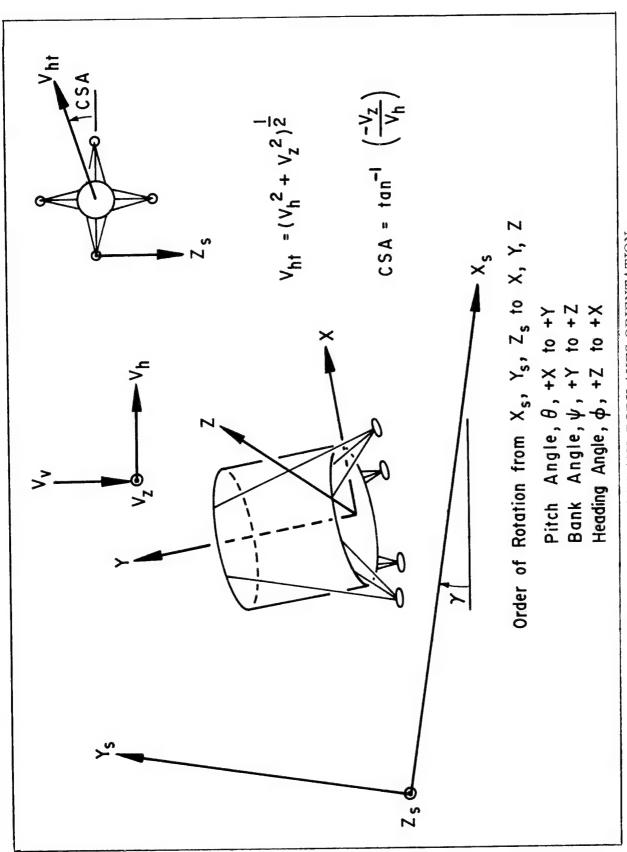


FIGURE 7. SURFACE AND BODY AXES ORIENTATION

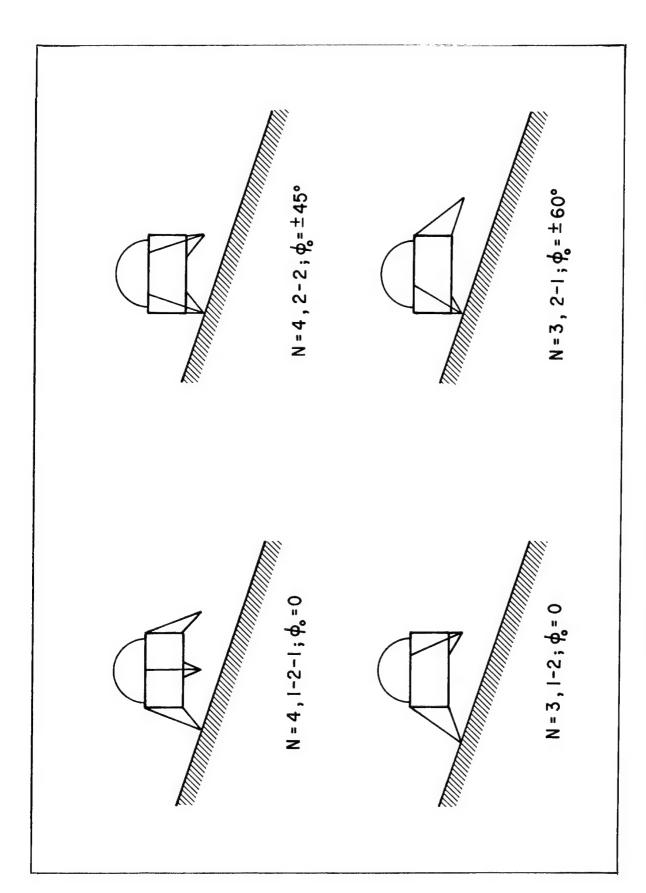
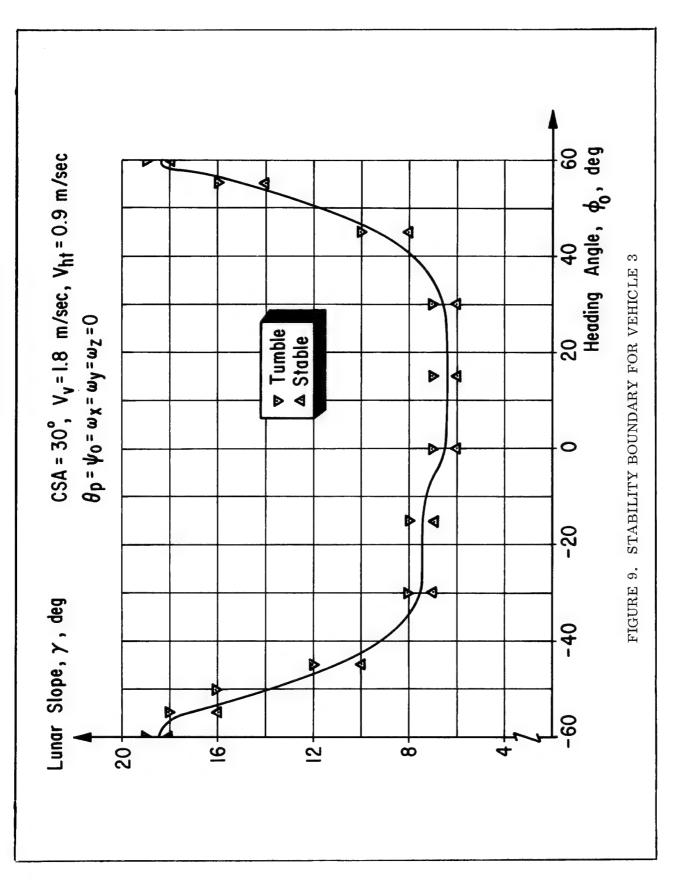


FIGURE 8. SYMMETRIC HEADING ORIENTATIONS



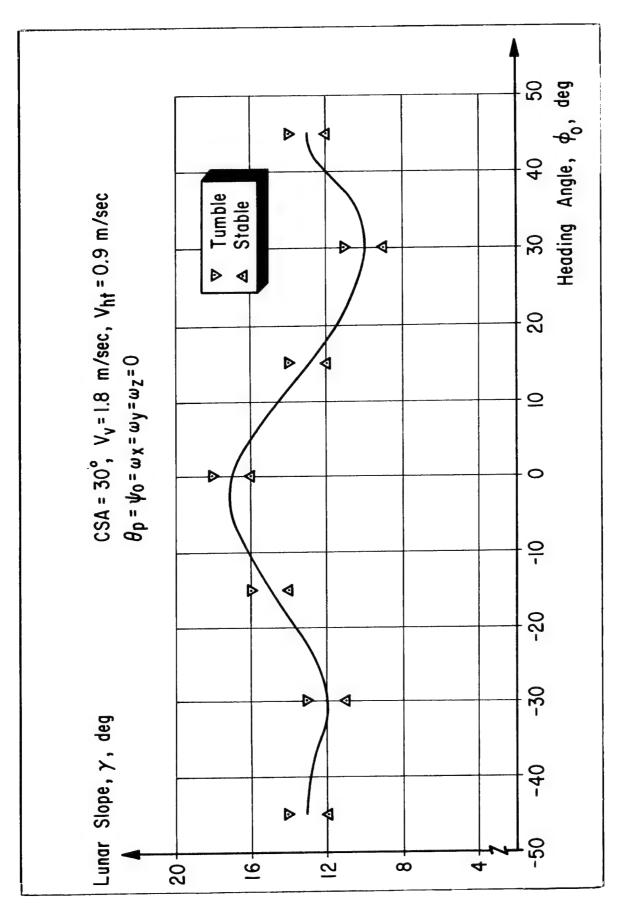


FIGURE 10. STABILITY BOUNDARY FOR VEHICLE 7

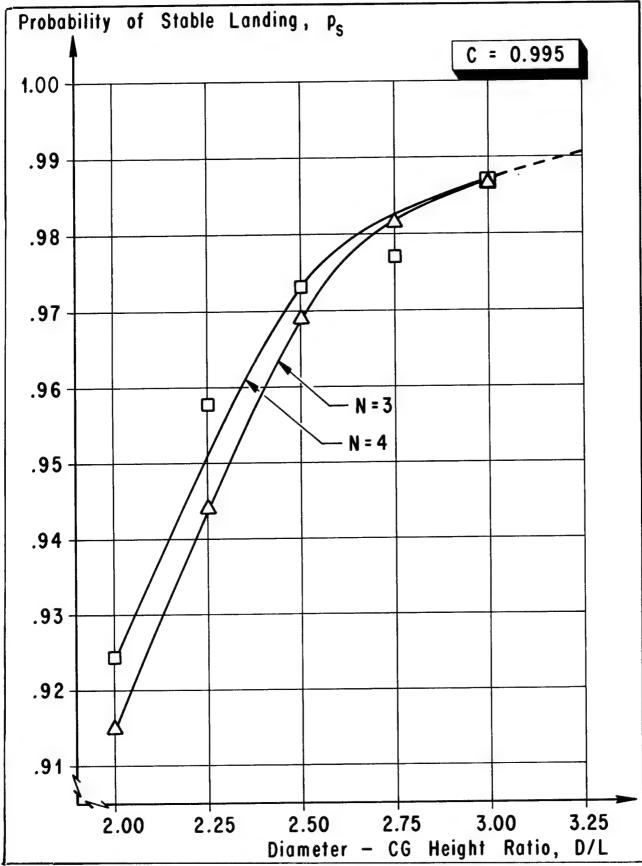


FIGURE 11. LOWER LIMIT PROBABILITY OF STABLE LANDING

TABLE 1. GEOMETRIC PROPERTIES OF LFV, LEM, AND LLS

Vehicle	D	L	Ya	Yp	r	k_z
	cm (in)	cm (in)	cm (in)	cm (in)	cm (in)	cm (in)
LFV	279 (110)	112 (44)	74 (29)	30 (12)	56 (22)	58 (23)
LEM	813 (320)	320 (126)	170 (67)	119 (47)	208 (82)	165 (65)
LLS	1966 (774)	605 (238)	381 (150)	180 (71)	330 (130)	259 (102)
Vehicle	D/L	Y_a/L	Y_p/L	r /L	k_z/L	
LFV	2.50	0.66	0.27	0.50	0.52	
LEM	2.54	0.53	0.37	0.65	0.52	
LLS	3.25	0.63	0.30	0.55	0.43	

TABLE 2. GEOMETRIC PROPERTIES OF VEHICLES I, II, AND III

Vehicle	Type	Y _a /L	Y _p /L	r /L	$k_z^{}/L$
I	LFV	0.61	0.31	0.57	0.48
II	LEM	0.61	0.31	0.57	0.48
III	LLS	0.61	0.31	0.57	0.48
Vehicle	Type	Y _a cm (in)	Y _p cm (in)	r cm (in)	k cm (in)
I	LFV		35 (14)	64 (25)	54 (21)
II	LEM	195 (77)	99 (39)	182 (72)	153 (60)
III	LLS	369 (145)	188 (74)	345 (136)	290 (114)

TABLE 3. LIMITING FORCES FOR TYPE I VEHICLES

Vehicle	N	D/L	C_{mp}		C_{mp} C_{sp}		$k_{ m V}$	
}			N	Ip	N	lb	N/cm	lb/in
1	3	2.00	3576	804	1192	268	8756	5000
2	3	2.25	3897	876	1299	292	7425	4240
3	3	2.50	4218	948	1406	316	6252	3570
4	3	3.00	4977	1119	1659	373	4588	2620
5	4	2.00	2682	603	894	201	8756	5000
6	4	2.25	2922	657	974	219	7425	4240
7	4	2.50	3162	711	1054	237	6252	3570
8	4	3.00	3738	840	1246	280	4588	2620

TABLE 4. DYNAMIC SCALING FACTORS

		Vehic	le Type
Quantity	Scale Factor	п	III
Acceleration	$g/g_{I} = \alpha$	1.00	1.00
Mass	$m/m_{I} = \beta$	14.00	42.00
Length	. $L/L_{I} = \gamma$	2.86	5.41
Angular Acceleration	a/γ	0.35	0.18
Time	$(\gamma/\alpha)^{1/2}$	1.69	2.33
Angular Velocity	$(\alpha/\gamma)^{1/2}$	0.59	0.43
Angle	(α/γ) ⁰	1.00	1.00
Velocity	(α·γ) ^{1/2}	1.69	2.33
Force	α • β	14.00	42.00
Mass Density	β/γ^3	0.60	0.27
Moment of Inertia	β·γ²	115	1229
Stress	$\alpha \cdot \beta/\gamma^2$	1.71	1.44
Spring Rate	α • β/γ	4.90	7.76

TABLE 5. INITIAL CONDITIONS FROM CHI-SQUARE DISTRIBUTION

, 									
F (x)	%	20.000	50.000	80.000	90.000	95.000	99.000	99.865	99.997
X	-	1.005	2.366	4.642	6.251	7.815	11.341	16.022	21.000
\overline{V}_{ht}	-	0.061	0.143	0.281	0.378	0.473	0.686	0.970	1.271
\overline{V}_{v}	-	0.122	0.286	0.562	0.756	0.946	1.372	1.939	2.541
γ	deg	2.1	5.0*	9.8	13.2	16.5	24.0	33.9	44.4
V _{ht} , I	m/s	0.057	0.133	0.261	0.353	0.441	0.637	0.901	1.181
V _{ht,II}	m/s	0.096	0.224	0.442	0.594	0.744	1.078	1.524*	1.996
V _{ht, III}	m/s	0.133	0.310	0.608	0.819	1.024	1.486	2.100	2.752
V _v , I	m/s	0.114	0.266	0.522	0.706	0.882	1.274	1.802	2.362
V_{V} , II	m/s	0.192	0.448	0.884	1.188	1.488	2.156	3.048*	3.992
$V_{_{ m V}}$, III	m/s	0.266	0.620	1.216	1.638	2.048	2.972	4.200	5.504

^{*}Values used to establish multiplication constants.

TABLE 6. TABULATION OF DISTRIBUTION FUNCTIONS

_	1 Distribution 0, σ = 1)	Chi-Squa	re Distribution (ν = 3)
F(X	X) X	F (X)	х
0.0000	-5.00	0	0
0.0000	3 -4.00	0.08111	0.50
0.0002	-3.50	0.19875	1.00
0.0013	- 3.00	0.31773	1.50
0.0062	-2.50	0.42759	2.00
0.0227	5 - 2.00	0.52444	2.50
0.0400	6 -1.75	0.60837	3.00
0.0668	-1.50	0.67900	3.50
0.1056	- 1. 25	0.73854	4.00
0.1586	-1.00	0.82820	5.00
0.2266	- 0.75	0.88839	6.00
0.3085	-0.50	0.92810	7.00
0.4012	- 0. 25	0.95399	8.00
0.5000	0 0	0.97071	9.00
0.5987	0.25	0.98143	10.00
0.6914	0.50	0.99262	12.00
0.7733	0.75	0.99818	15.00
0.8413	1.00	0.99956	18.00
0.8943	35 1.25	0.99997	21.00
0.9331	.9 1.50	1.00000	24.00
0.9599	1.75		
0.9772	2.00		
0.9937	79 2.50		
0.9986	3.00		
0.9997	76 3.50		
0.9999	97 4.00		
1.0000	5.00		

TABLE 7. SUMMARY OF TOUCHDOWN DYNAMICS RUNS FOR VEHICLE 1

Gro	oup A		Gro	up B		Grou	ір С	
Case	Result	γ , deg.	Case	Result	γ , deg.	Case	Result	γ , deg
6	S	13.0	145	S	10.4	13	S	14.8
8	S	10.3	168	S	12.7	61	S	9.5
9	S	11.8	172	S	1.7	95	S	8.4
14	S	14.2	235	S	10.0	100	S	14.3
39	S	15.7	238	S	9.7	102	S	13.9
43	${f T}$	6.9	242	S	12.4	103	S	4.9
46	S	8.9	247	S	8.4	123	S	8.0
64	S	12.4	250	S	8.0	243	S	12.2
66	S	12.8	259	S	6.8	337	S	8.3
82	S	15.3	275	S	13.4			
84	T	15.5	277	S	15.0			
87	${ m T}$	11.5	288	S	12.4			
118	${ m T}$	10.7	289	S	8.8			
120	S	10.3	291	S	7.2			
144	S	14.1	296	S	8.6			
148	S	11.8	303	S	9.5			
157	${ m T}$	23.0	307	S	14.8			
158	S	16.0	309	S	10.7			
173	${f T}$	15.5	314	S	13.0			
177	T	19.3	315	S	12.1			
179	S	10.4	319	S	7.3			
187	S	9.1	321	S	13.5			
198	S	14.9	327	T	14.9			
206.	S	16.8	338	S	6.9			
207	S	16.9	341	S	8.4			
211	S	11.3	357	S	15.3			
212	T	9.7	363	Τ	14.0			
214	S	11.3	364	S	15.4			

TABLE 7 (CONTINUED)

Gro	oup A		Group	В	
Case	Result	γ , deg.	Case I	Result	γ , deg.
215	S	17.7	368	S	14.1
234	S	14.4	369	T	12.5
294	S	16.4	386	S	9.5
299	S	11.6	396	S	13.3
328	S	17.7	397	S	8.7
332	S	19.2	400	S	6.7
333	S	16.3			
339	T	17.0			
345	S	17.2			
346	${f T}$	21.8	Group A:	Cases	s which were thought to have a
350	${f T}$	20.6		subst	antial probability of tumbling.
351	${f T}$	16.5	Group B:	Cases	s which were believed, with
371	${f T}$	21.2	-		antial uncertainty, would be
379	S	14.7		stabl	•
389	S	10.9	Group C:	Case	es which were thought, with only
390	S	18.3	*		all degree of uncertainty, would
398	${f T}$	23.5			stable.

TABLE 8. SUMMARY OF TOUCHDOWN DYNAMICS RUNS FOR VEHICLE 5

	Gro	up A		Gr	oup B		C	Group C	
_	Case	Result	γ , deg.	Case	Result	γ , deg.	Case	Result	γ , deg.
	14	S	14.2	6	S	13.0	13	S	14.8
	39	S	15.7	9	S	11.8	64	S	12.4
	66	S	12.8	43	S	6.9	87	S	11.5
	84	S	15.5	46	S	8.9	100	S	14.3
1	57	T	23.0	61	S	9.5	102	S	13.9
1	73	${f T}$	15.5	82	S	15.3	118	S	10.7
1	77	S	19.3	95	S	8.4	145	S	10.4
1	98	S	14.9	103	S	4.9	148	S	11.8
2	06	T	16.8	123	S	8.0	275	S	13.4
2	15	S	17.7	144	S	14.1	303	S	9.5
2	89	S	8.8	158	S	16.0	321	S	13.5
2	96	S	8.6	207	T	16.9	338	S	6.9
3	15	S	12.1	212	S	9.7	389	S	10.9
3	19	S	7.3	228	S	9.3	396	S	13.3
3	28	Т	17.7	234	S	14.4			
3	33	S	16.3	238	S	9.7			
3	39	${f T}$	17.0	270	S	6.7			
3	46	T	21.8	277	S	15.0			
3	50	T	20.6	288	S	12.4			
3	51	T	16.5	291	S	7.2			
3	64	S	15.4	294	Т	16.4			
1	68	S	14.1	307	S	14.8	Group A:	Cases whi	ch were
1	71	T	21.2	314	S	13.0	010ap 11.	thought to	have a
	90	T	18.3	327	S	14.9	,	substantia bility of to	
3	98	Т	23.5	332	Т	19.2			
				337 345	S S	8.3	Group B:	Cases whi believed,	
						17.2		stantial ur	ncertainty,
				357	S	15.3 14.0		would be s	stable
				363	S		Group C:	Cases whi	
				369	S			thought, was a small de	
				376	S	5.4		uncertaint	
				379	S	14.7		be stable	

TABLE 9. SUMMARY OF CASES WHICH TUMBLE

Case				Vehi	cle			
	1	2	3	4	5	6	.7	8
36	${ m T}$	Т	T		T	${f T}$		
.43	Т							
84	\dot{T}	T						
87	Т							
118	Т							
129	T	${f T}$			T	${f T}$	T	
157	${f T}$	${f T}$	${f T}$		${f T}$	${f T}$		
173	T	${f T}$			T			
177	T							
206					T			
207					${f T}$			
212	T							
294					\mathbf{T}			
317	T	${f T}$	${f T}$		${ m T}$	Τ	T	
327	\mathbf{T}							
328					T			
332					T			
339	\mathbf{T}	${ m T}$			\mathbf{T}			
346	\mathbf{T}	T			\mathbf{T}			
350	${f T}$	${f T}$			${ m T}$	Ť		
351	T				T			
363	${f T}$							
369	Т							
371	T	${ m T}$	T		${ m T}$	T		
390					T	T.	an.	
398	T	T			T	<u>T</u>	T	
n _f	20	11	4	0	17	7	3	0

TABLE 10. SAMPLE PROBABILITY OF STABLE LANDING

		p _s	= (400 -	n _f) /400				
Vehicle	1	2	3	4	5	6	7	8
p _s	0.9500	0.9725	0.9900	1.0000	0.9575	0.9825	0.9925	1.0000

TABLE 11. LOWER LIMIT PROBABILITY OF STABLE LANDING

	C = 0.	950						
Vehicle	1	2	3	4	5	6	7	8
p _s (Ref 23)	0.9282	0.9549	0.9773	0.9925	0.9369	0.9674	0.9807	0.9925
p _s (Ref 24)	0.9303	0.9575	0.9787	0.9925	0.9394	0.9696	0.9817	0.9925
	C = 0.9	995						
Vehicle	1	2	3	4	5	6	7	8
p _s (Ref 23)	0.9149	0.9439	0.9689	0.9868	0.9243	0.9577	0.9728	0.9868
p _s (Ref 24)	0.9223	0.9498	0.9713	0.9868	0.9314	0.9620	0.9744	0.9868
					· · · · · · · · · · · · · · · · · · ·			

APPENDIX A TOUCHDOWN CONDITIONS

MONTE SARLO D-TERMINATINA DE TOUCHDOMA COONTIONS, SOCIASOR LAVENDER, DESGRAMMED CAPPS, JOH 551120.

THA	035751	9.	ο , ο ,	D 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0		2.7	11.4	10.8	2.1	16.4	1.1	4.8	5.7	11.8	5.8	7.1	1.4	21.1	2.5	12.7	14.1	3.4	1.6	3.4	14.8	13,3	2.7	# C	1.7	7.7	4.9	13.8	6.4	7.7	1.8	1.1	2.8	13.9	15.4	٠, د د	7.6		17.6	,,,	43.7	9.	7.5	6.2	12.9	6.1	2.5	20.9	2.4
74	753/5	-2.1	-4.7	n io		-2.2	2.6-	6.0	5.5	٠ ٢٠٢	17.5	-3.1	4.6	-2.2	3.5	ر ب	10.4	-4.7	5.1	1.0	101	-3.7	r. r.	4.3	-2.3	11.0	α. 	14.04	F -	2.6	ر ار ار	7.6	-1.6	٦. د	1.3	1.6	1.0	-9.3	-2.7	. o	0.0				4.7	2.0	2.0-	-4.3	7.2	٠. ص	0 - 1	-1.2	1 00
> 3	S/9:u	ص د	13.0	× ×	0.0	4.0-	œ.	4.6-	0.1	5.9	2.7	-3.8	1.5	1.7	6.6-	4.3	-3.0	1.5	.2	-5.3	2.0	3.5	9.0-	-1.2	0.9	1. 1	00	00	1.4	4.1	0.0	9.0	4.9	2.1	5.0	0.3	0.0	90	9.6	 	0.5	ا د د	0 10	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1		101	1.6	-1.1	-1.0	1.0	4.6	11.4	11.8
× 7	NE3/6	٦ 4 • ا	~ r ~ r	ر د د ا		1.5	1.9	1.7.	9.6	-1.1	6.3	0.1	4.1	-1.0	-0.4	0.2	0.0-	-1.0	-3,3	1.8	-1.4	1,4	-8.0	-2.3	-1.6	-1.8	53.3	0.0	0.0-	-5.4	-0.4	α +1	4.1	-6.1	6.0	2.7	9.0	-2.0		1.2.	\ . ·	1 · ·		4 0	4	4	-0.3	3.1	-2.6	0.1	7.01	-1.8	7.7
7 1 4	5.4¢	29.5	6.0	0.0	-28.7	-8.3	-28.1	2.1	-7.8	32,3	32.2	6.4-	1.9	-28.4	8 . K	-20.5	n.3	-10.3	29.0	28.2	-12.6	-3.5	-34.5	-15.6	-36.7	-7.9	-19.2	19.4	17.9	-23.6	-15.2	43.1	-28.7	39.5	-22.5	-34.5	ທີ່	7.3	9.04	106.00	24.0	1 1 7	2 2 7 7 8	3.4	00	-21.7	25.4	32.4	-0.7	16.1	19.5	6.1	42.6
PHI	9=4	-3.) to	7	0.0	128.7	51.7	31.9	32.1	55.5	-27.7	32.2	6.1-	-28.1	-28.4	α.	30.8	-20.7	49.7	29.0	28.8	47.4	-33.5	25.5	-15.6	-36.7	-37.9	-19.2	19.4	-42.1	36.4	14.8	13.1	31.3	-50.5	7.5	-34.5	-56.5	-58./	7.0	9 . 4	0.40		2 2 4 =	4.0	41.2	-21.7	55.4	32.4	29.3	46.1	19.5	-23.9	-47.4
PSI	DEG	9.8	\ .) (\ • • • =	٠° ٠ ٥	2.0-	5.41	3.00	e-1 67	4.7	# 0 · B	0 . 4	1.1	1.5	0.7	-0.3	-2.4	1.0	-2.1	U * 0	1.5	2.3	2.4	1.6	0.4	-1.	z * ?	-1.1	8.0	2.4	4.8	0.4-	5.9	1.7	-0.5	3.8	7 0 - 7	ц. С.	4 0	2 0 1				2	~ 0	2.4	-2.4	-1.4	-1.4	3.0	9.0	-1.9
GAMWA	е Ц:	α!	c (1 / L		1 4.0	α.	17.4	11.0	U • >	4 • 4	2.2	14.R	14.2	∪ * }	α 	5.2	а О	7 • 5		π,	10.0	· · ·	1.0	4. 4		م	7. 7.	α.~	4.1	, c	α.	ດ ຕັ	٠,٠	а п	24.3	14.2	4	15.7	= L		0 4	, c	· ~	0	5.9	۳. 0	3.6	4.	K	4 1	1.7	1.0
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115	Jak/aI	3.5	N 10	9 11		4-4 00	7.4	-13.7	2.1	-12.5	5.9-	3.9	6.4	7.8	-1.6	-4.5	1.3	1.6.1	5.4	6	-12.0	-3.3	-1.4	2.5	14.7		2.1	-0.1	4.	7.4	6.4	13.5	5.0	-7.6	1.1	-0.1	2.7			 U	1	C +	- 1		-41.8	0.7	7.5	5.9	12.9	00.0	0.0	-11.6	2.3
4 N	ปรร/\i	, i	L .	د اد اد		۲. ۲	-8.7	6.1	-1,1	17.4	-1.1	. s	n . 8 −	σ,	15.6	α΄	7.5	-7.7	7.1-	12.7	7.5	٥. ٨	n. 7	3.4	-1,	٥, ٩,	1.7	c . c 1	-1.7	12.1	4,5	-3.1	-4.1	-1.2	1.3	-n.1	٥, ۶	τ. α.	ا وأني	ۍ . ت. د ا	- 0	, ,		, ,	-12	-1.4	-0.4	ec .	9.0-	ا ا ا	- 0	-17.3	0.7
۸۸	1 ./ SF.	5.5	¢ ,	ی م د آج		14.3	4.4	14.2	39.2	4.2	6.2	2.9	4.4	. 35.3	13.5	3.0	13.3	32,5	3.6	5.6	5.5	3,3	20.5	12,3	3.0	12.0	26.5	2.6	4.6	4.5	2.2	24.0	10,8	1.6	10.4	21.8	6.0	8		N .	6 c	0.60	2 4	17.6	34.5	1.3	1.0	29.6	15.6	7.0	0.6	14.1	23.4
Pulo	0=0	2007	7 5	100	531.3	174.7	151.3	172.1	172.2	212,3	32.2	(52,1	6.16	331.6	\$51.2	150.3	90.3	169.7	50.0	24.2	167.4	84.5	145.5	544.4	123.3	82.1	540.3	10.4	197.9	154.4	254.8	133.1	151.3	5005	247.5	325.5	183.5	181.5	510.1	000	1740		226.7	124.9	161.2	33R.3	295.4	32.4	269.3	200	10.5	96.1	312.6
184	Ç11	133.7	4.5.4	- u	~	6.1.5	5.8.6	43.7	746.8	0.77	122.R	7.416	7.43.5	2,1,2	14.8.9	45.4	234.9	111.3	253.7	6.403	78.1	C. D. F	6.54	4.1	236.0	1.7.8	3.9.5	91.1	132.7	254.1	215.5	255.9	198.1	66.3	320.3	141.3	23.2.3	15.4	5.00	£ 4	7.4.	226.3	4 4	326.0	117.1	217.2	247.3	247.3	247.2	0.	3,50.3	146.1	285.6
>	3.48	2 4 6			0.017	370.0	9.512	$\overline{}$	\subset	C	C	C.	C.	0.00	C	-	C	=	C.	0. \$47	\subset	=	~	(T	A . C	=	0 •	\subset	ր, դ	0		0.3	<u>.</u>	ċ	ก 🕯 ป		- 1	, .				. 4		0.0		٦.	0.2		5.3	را د ا	0.04	0.5	
> >	nr :	9.124	:: F F F F	1.1.0	1.157	n. 391	0.110	0.330	1.072	1.115	0.171	0.140	0.111	P.953	0.350	n.135	1.35.1	αεκ·υ	0.03A	1.151	n.15n	0.090	n.797	n.337	0.032	0.329	0.725	0.072	0.124	0.122	0.051	N. 654	0.295	n.044	n.285	n.595	0.024	0.030	45.05	0 10 10 10 10 10 10 10 10 10 10 10 10 10	7000		234	7.4	n.942	N.034	n.02¤	0.810	0.427	0.192	0.179	0.386	0.63A
CASE	0 7	e-1 (V P	c 4	ιc	r	7	αr	Э	10	11	12	13	14	15	16	17	o r T	19	20	2.1	2.5	23	24	52	56	27	2 ₽	50	3.0	31	32	33	3.4	35	36	37	. c.x) + (4.7	. 4	4	. 7	4.	47	4.9	40	0.0	12	0 1 0	54	55

L.	7	σ.	! ! <u>c</u>	v =	~	0	6	23	ao 1	٠.	! 		0 0		U 4		. 0		+ 0			0 4	0 0		v c	• •	1 <	. 11	. 0	-		0	0	. 7	0.	2		o i	• !	v. c	•	-	10		2 14) -	10		5	90	ю. С	7
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N 2 3 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1 2 1	14.2	6.	+11	0.0	4	٦.6	œ.	٥.2	-1.1	-5.0	60	-3.6	۳. ا	- 0				0	- F	ola Sia		2 1	4	-3.7	7.5	7.1	0 0				, ,		-	80	-4.3	4. 80	-1.2	2.0	2.0	,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,	2	- 4	2		4 .0	- 4	c +	4 6	1.6	80	## F	-
× A Gud	0.7	-1.1	-0-3	2.4	7.0-	2.7	3.6	-1.0	-2.3	4.0-	4	-4.1	-0.5	- L	C.U.	0 7		21.5		2	0.1	ر. د ر	5.5	0.0-	4.4	5.0	00 00	10.0		- H	0 0 0	4.4	00.0	4.0	3.5	6.0	1.8	-1.9	-3.0	910	-3.7	× × × ×	9.0	0.0		- F - 7	11.		0.7	-2.7	-3.4	-0-
XX	6.0	-2.8	4.0	7.4	-3.2	. w.	4.1	1.2	-1.4	2.5	-2.0	0.2	5.0		٦. ١	•	> · · ·	•	H	2 0	6.2	-1.0		-0.7	6.1	4.0	0.7	2.5	1.0	4.7	. 4	.5.7	3.0	0.7	-2.0	1.7	-2.9		0.7	1.5	6.0	200	N (0 t		e c	200	7.1.7	-2.0	4	2.3	0.0-
PHT 4	39.1	14.6	1.7	1.52.	0 0	-14.0	-28.1	7.7	4.5	-40.9	-35.3	20.2	35.6	100	000	41.4	100	-38.5	0 4	11.0	6.55	2.6	54.5	-10.0	-25.5	00	2.5	42.5	20 m	12.	7.6	27.5	11.3	-45.0	38.6	-7.8	4.4	-41.0	-27.7	35.6	-31.3	4. 4. 50 (0	15.2	27.20	20.5	2 2 2 4 2	0.41.	100	4.4	-24.1	8.2	0.0
PHI 3	-50.0	-34.6	1.7), v	0	-14.0	1.9	37.7	-26.6	49.1	24.7	20.2	35.6	140.1	7.0	238.0	000	51.5	553.0	1 1 2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	36.1	0 1	-34.5	-40.0	-25.5	0° (56.00	17.5	4 1	14.0	1 1 2 4 5	22.5	4 4 4	15.0	8.6	22.2	55.6	-11.0	-57.7	35.6	28./	41.00	7.04-		-39.5	50.00	0.00	0.00	23.6	-24.1	-51.8	-59.5
PSI	0.7	3.9	-	0 0	7 2	4.4	1.0	-1.3	1.1	-3.4	-1.2	· 0 · 5	5.0-	1	-2.8	1.4	-1-	1.5	2.5	6	-0.3	0.0	# 0 · 8	-2.3	1.7	-0.7	1.7	-2.1	-0.7	0			0 .	0.0	1.9	-2+0	9.0-	0.0-	0.0	9.0-	-1.9	C .	9.0	2.0	1.9	9.0-	0.0	0 4		14	9.0-	1.9
GAMMA	5 6 2 6 2 7	1.1	0	7.1	. u		4.6	12.4	7.6	12.8			10.4	5.0	c ac	رب د	œ !	4.7	6.		ec ac	٠.	11.2	4.9	5.2	15.3	α ir	15.5	5.5	0 1	11.5	C 0	0.0		0		2.7	α.	6.2	11.1	4 B	ر. 1.	14.3	α.	13.9	0.	4.6	າ. ວ່າ	1.0	8.0	1.5	4.3
THETA	5 - 1 - 3	6.0-	1.4	ις. (C. (C. (C. (C. (C. (C. (C. (C. (C. (C.	7.0	2 M 2	α	0.5		6.01	-1.5	0.5	-1.3	-0.4	۲۷. د		12.8	1.3	1.7	-1.4	-2.9	-1.1	,	2.5	-1.2	-3.3	-1.6	1.4	1.1	-3.8	⁶ 1 (δ, ε	ار م			- 2	-1.5	α.	-0.1	1.1.0	-1.1	-1.2	, n. 8	-0.3	0.5	-2.2	2.0	F. 0.5	2.5	0 -	0.4	2.1	0.0
17	1 N / SEC - 5 - 1	-12.9	-3.0	1.2	-110	12.0	1	-15.2	0.2	6.0	2.1	9.6	6.0	-5.9	0.2	-10.5	4	-0.5	e-1	-1.9	-15.9	0.5	8.1	-7.5	0.5	-8	4.8	2.7	80 1	3.7	-1.7	2	11.0	0	4.	-1.0	5.6-	4.0-	15.4	1.4	13.9	8.1	\Box	_	₩.	2.0-	(0)	0 · d	D 10	-0.7	9	2.7
7 >	11 000	1.3	7.0-	0.1	2	^ q		-2.1	8.01	-1.6	0.3	-3.1	6.4-	0.1	0.1	-7.3	-0-2	9.9	6.3	0.4	11.0	8.	-2.0	-2.5	0.1	Մ.	-13.6	3.5	-2.0	-5.9	α : ()	C .	. u	1,0	1 70 1		10.0	9.6	8.4	-5.3	3.2	5.7-	6.1	-1.2	-2.0	1 1 3	-3.4		1.6	4 0	. b. 3	-1.6
^^	IN/SEC	31.4	21.1	12.3	0.	1.61		17.1	21.7	24.0	15.5	6.5	2.4	14.0	1.7	7.9	12.7	15,5	15.0	11,3	6.2	33.7	10.1	28.8	4.9	6.8	11.0	10.6	7.7	3.1	10.6	9 1	17.7	1 to		4	4	34.5	12.9	3.1	11.8	24.6	1.7	3.5	3.1	56.8	18.2	60	20.0	14.7	23.9	32.9
PH10	200	85.4	٠ 🛶	57.9	254.0	230.1	0 4 4 0	277.7	93.4	40.1	144.7	20.5	35.6	190.9	126.2		56.5		1.86.4	101.3	156.1	350.8	325.5	80.0	334.5	A.9	183.2	137.5	231.6	105.7	119.7	273.7	207.5	2010	100.00	0.690	175.6	229.0	62.3	35.6	148.7	41.8	74.8	247.7	200.5	293.3	166.0	178.6	331.1	335.9	188.2	180.5
CSA	المار 10 مار 10 مار	34.4	1.13.7	82.9	22.0	731.4	140.0	5.8.9	106.5	4.4.0	273.7	0.50	1	49.0	m	1,75.5	9	•	59.6	77.4	55.2	153.0	250.6	178.2	245.7	63.1	150.4	217,7	234.8	211.9	148.9	45.9	262.8	7.5	710.7	• •	27.70	2	238.6	194.8	51.0	227.1	3.4	99.0	154.9	170.6	146.3	81.9	337.5	0 2 6 F	143.6	238.8
, ,	SAR AAR	354	.084	033	.081		- 0.		122	71	, ; _c	7.75	134	1.161	0.004	0.344	0.032	0.241	0.257	0.053	0.52A	A 048	n.235	0.217	0.015	0.277	1.394	0.121	0.095	0.180	0.080	0.100	0.318	0.219	10.40	140		0.267	0.470	0.150	-	0.303	16	.21	. 06	.03	11	. 02	C	0.179	21	. 08
۸۸	U 4 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	N. 71/	1.575	n.336	0.134	0.521	0.120	7627	י בי	1000	404		0.056	0.394	0.046	0.215	6.346	0.423	0.400	0.309	0.170	1.921	0.276	0.787	n.134	242	0.300	0.28A	0.211	0.086	n.537	0.179	0.484	0.030	0 + 1 + 0	200	11. E3.	0.942	0.352	0.095	0.321	0.673	0.044	0.034	0.085	1.551	0.498	0.223	0.836	197	n.654	0.900
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THETA	0 • 3	-0.4	0.4	2 0	0.0	-1.3	5.0-	3.0	2.1	200	2.0	2 0	-1.5	0.2	-1.9	-1.7	9.0	4 4	-	1.0		-1.1	4.0-	3.4	2.3	-1.9	C 10	4.0-	-1.2	0.1	-2.0	ן לין כ		0.2	-1.9	-2.1	0.0	9.0-	2.7	00.	-2.4	τ.	2.4			-2.B	-2.5	U · T
2 /	5.1	2.9	0.7	10.1	-2.9	31.0	6.0	-1.2		1:5	4. 0	2 4	7.7	13.4	-4.8	-6.2	-17.9	0 -	4		00	-3.1	-4.2	0.0	5.6	-0.8	-11.7	2	0	-3.3	10.2	19.1	0 0	1.0	-8.2	9.1	0 + 0	13.6		1.2	4.8	9.	-2.3	4.4	7.9	0.9	2.1.	C.4.
H >	IN/SEC	-0.7	-4.6	0 M	-0.5	-25.9	1.1	6.5	47 (9.5	or o	- 2	7 -	15.9	4.1	-1.0	-6.0	, t			0.7	5.3	-3.9	4.0-	-14.5	-1.9	¢. c	- I	. 60	-2.0	-18.4	4.4			7.2	-0.3	د ر ار د	200	. P	C.	0,0	4.4	11	C P	. 6	3.0	2,5	
>	1 × SEC 2.9	24.5	10.3	0 4	16.7	27.4	39.3	31,4	19.8	11.0	د د د د	12.0	7.6	11.6	13.5	12.4	8.8	3.7	4 4	0 4	5.7	4	5.3	£.5	1.4	20.1	e 5	000	13.5	20.4	24.5	21.6	4	0.0	11.1	30.8	4.0	0 0	, 00	5.4	46.0	13.2	2.8	10.7	39.7	6.0	47.6	45.U
PH10	219.8	267.6	95.2	170	57.8	85.2	252.5	199.7	286.8	153.9	100.0	34.6	301.3	148.0	134.6	261.1	167.6	213.9	7	110.5	558.5	24.5	190.4	136.2	221.9	87.6	93.2	164.1	229.4	74.7	50.0	185.0	1740	319.8	284.6	29.3	274.0	103.0	47.4	131.7	356.0	0.1	144.2	68°5	536.0	319.8	83.5	347.1
CSA	286.6	257.4	188.1	78.7	99.7	230.1	320.4	10.6	20.7	350.8	230	1/0.7	100.3	320.0	49.6	99.1	108.5	6.77	2 4 7 3 0	000	\ie	. 10	4	141.2	190.0	158.6	87.2	184	352.5	120.8	218.9	257.1	23201	160.0	48.7	256.5	44.1	270.2	326.6	3 53.9	341.2	228.3	115.4	3.22.4	256.2	345.0	29.7	56.4
7	94R	0.082	1.128	0.001	0.081	1.103	n.037	Π.181	0.142	0.262	0.123	0.130	0.254	0.56A	0.172	0.171	0.514	0.241	2000	0.140	0.231	0.123	0.154	0.012	0.403	0.057	0.320	2000	0.050	0.104	0.575	0.535	0.00	2 6	29	25	0 0	\' ~	, -	4 0	4-4	=	<u>د ا</u>	7, 4	-1 Cı	1.084	80	0
. >>	7 A 7 0	0.659	n.291	0.006	0.454	n.748	1.074	0.85A.	1,541	0.300	0.095	0.411	7000	318	0.370	0.340	0.241	0.102	0.221	0.174	1 77 B	0.110	0.146	0.124	0.039	n.549	0.233	10000	0.450	n.55R	0.670	0.589	0 4 UR	0.010	0.304	n.840	0.129	0 250	0.00	9.14	1.25R	0.350	n.077	0.233	1.096	0.025	1.300	0.602
CASE	10 P	167	168	169	171	172	173	174	175	176	177	1/8	4 t	100	182	183	184	182	000	\ 0 0 1 1	0.00	000	191	192	193	194	195	100	100	199	200	201	202	200	205	206	207	200	, ,	211	212	213	214	215	217	218	210	220

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DEG/S	FT F		6.1.	9.4		1.4	-1.5	-1.7	0.0	10.9	0.1	4.4	4 .	\ 	, ,		. U .	0.0	-1.7	-1.4	1.5	0.1	1.5	 	\ O	0	0.1	-5.0	4.4	-1.8	4 R		-1:1-	9.0	-2.0	0 +	1 P . C -	1.1	-2.0	N .	 	10.01	4.5	5.9	-2.7	80 H	ر د د د د	0
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ກ (D ເ ທີ່ກີ ພິດກິ	200) .V	0.7	∵ α N ⊏	130	-0.7	S 0	œ ·	9 0	10.	-1.5					9.0-											6.0										5.0	-1.5	7.0-	- C		-3.4	6.0	-1.7	0.5	2 0	2	:
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13 SEC	4.0	15.5	2.7	1	12.9	11.6	w	N F	5° L	ນ ເ ວິດ	13.8		4 M) M	30.2	14.5	5.6	19.0	3.6	0.6	13.2	15.0	20.0	V	19.7	5.9	15.2	34.2	80.	ა ი ა ი	34,4	15.1	5.9	19.6	200	13.2	14.9	13,1	2, 4	0 00	5 5	14.4	29.7	2.2	2,5	0.70	13.6	
05.0	214.1	177.5	7.007	187.2	354.3	293.3	15.6	0.000	1 C C C C C C C C C C C C C C C C C C C	24.7	7.00	212.1	194.4	316.7	218.9	261.1	8.3 . 1	45.1	147.0	ο α α	٠	212.3	235.9	600	98.2	239.4	160.6	721.3	62.8	2 4 4 5	65.5	106.2	284.4	247.4	7. V . V	248.7	48.9	349.1	78000	289.2	69	348.9	48.6	8.	227.8	. 4	86	
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GAMMA	6 6 8	α ۳.	10.2	7. 4	- α. > c	2.1	κ. α	17.0	7.0	α, 4	2.5	6.6	7 0	4		5.5			14.5	۳ ۲ ۲ c	÷ (, a	4	15.3	4.6	7.9	1.9	4.0	α .	- L	, tc	1.7	4.4	14°1	2.5	21.2	1.4	K . 7	1 1 1	7	. 4	0.0	14.7	A .	0	10.0	0.3	2.2
THETA	DEG	-4.5	ν, 4 τ	1 2	-2.5	0.5	10.0	-0.4	-0.1	2.8	1.3	3.5		3.5	1.1	2.3	-0.4	/•U-	- 1		9 4		2	1.7	2	-1.1	-2.5	6.0-	o^ o	7 -	3.1	-1.6	1.4	. 4 - 4 - 3	1.2	1.9	-1.0	-1.	. 4	-2	4.	-0.5	0.6	-1.6	-1.7	100	-0.0	-3.8
7.7	INISEC	-3.6	. n	200	9.0	14.0	7.6	-3.7	-3.0	4.0	-3.7	-0.5	o un	13.1	2.8	-8.4	0.2		6.0	0 -	1 0		-1.7	5.4	2.7	-4.1	80	4.6	7		-3.1	-3.9	4.0	-14.C	-6.0	-18.4	0.8	1.7.	200		-5.1	2.2	2.0	-0.5	-0.0	.0.0	1.7	0.0
z <	INISEC	3.4	7.1	4	4.6	1.3	117.0	15.8	6.5	5.0-	10.9	2.0	10.	5.7	4.4	-4.8	3.0	ر د د د د	2.9		2 4	4 4	- 0 -	. e	3.4	-10.n	4 1	3.6	- 6	5 4	-2.7	3.8	- 0	c c	12.3	-3.5	₩, 1 ₩, 1	r u	. 4	0	2.1	-2.4	4.7	0.4	c	, K	1.3	1.8
^^	INISEC	3,1	7.07	28.7	34.0	25.4	10°8	46.5	9.7	22.7	ر ال	2,5	, 4 , r.	4	16.9	6.5	20.1	90	0.0	12. Y	0 0	9	0.7	13.5	2.4	9.6	16.8	24.5	0 0	13.6	9.9	28.2	٥,٠	50.0	3.3	3.8	° 10	4.01	4 4 0 C	14.5	0,3	5.6	8.3	0.0	7.2	0.00	8.9	27.8
PHIO	DEG	83.4	1/8.5	67.8	222.5	157.0	95.0	81.2	214.5	128.6	182.7	150 7	104.5	354.3	32.0	205.6	~	00	120.4		7	298.8	1.8	4.7	S	ارات درانا	243.0	r a	1 4	0	5.5	257.4	0 4	173.6	L	7.3	. ; o	0 0	183.9	5.3	7	٥.	9.1	80.5	2 6	113.1	284.0	234.7
CSA	DEG	46.5	4 4 4 6 4	160.0	217.6	275.2	210.4	147.4	24.6	221.R	18.0	212 0	249.7	246.5	203.2	119.9	356.4	192.9	0.440	0.18	218.0	214.1	170.1	86.0	321.9	157.6	313.3	4 4 4 4	70.0	175.3	130.6	45.8	4460	271.0	55.9	100.8	155.5	2 4 20	359.3	233.8	68.1	222.4	3.46.6	50.7	2 0	12.7	306.5	180.2
× ×	3 A R	0.134	0.210	0.104	1.125	0.384	0.763	0.18a	0.195	0.017	0.515	0.177	0.163	0.391	0.197	1.264	0.083	0.050	0.100	0.070	0.220	0.15a	0.269	1.114	0.11a	1.294	0.182	0.100	0.00	9.204	0.112	0.140	0.404	0.065	0.374	0.512	0.11 A	0.141	0.030	0.041	0.157	ก. กลุ	0.139	ο το ο ο ο ο ο ο ο ο ο ο ο ο ο ο ο ο ο	000	0.027	0.059	0.225
^^	e €	0.034		0.735	0.930	0.694	0.222	1.271	1.254	0.620	0.057	0.154	0.123	0.012	0.453	0.177	0.000	0.0.0	324	0.351	0.296	0.186	0.019	0.358	0.056	0.251	0.40 A	0.726	0.579	0.372	0.191	0.771	430	1.650	0.099	0.105	0.053	0.350	0.109	0.397	0.00 e	1,153	1.227	Y 7 7 0	0.00%	0.639	242	0.750
CASE	C I	331	2 00	334	335	336	338	339	340	341	2 4 2	344	345	346	347	۵. د د د	240	354 154	352	353	354	355	356	357	353	259	000	362	363	364	365	366	368	369	370	371	374	374	375	376	377	378	6/5	⊃ × 00 ×	300	383	4 00 P	282

VHT	211	6.5	2.0	6.	00	6.	4	0	. 2	4.	. 0	.5	4.	9.	80.	6.
		- 1				-					i				١.	
25	156/8	٨. ٨	4.4	5.3	2.5	-2.7	N	-4.9	0,	5.0	e: 00	. S.		-0.7	2.1	-1.5
7.	DEG/S	4.6	4.5	-1.5	-3.4	-1.7	2.5	1.9	-3.7	2.3	3.6	2.0-	11.5	0.4	-2.1	-1.7
×	DEG/S	3.5	1.3	-1.0	.3.7	0.	1.4	3.9	4.00	-4.7	4.1	1.5	-0.8	4.2	-0.8	1.6
PH1 4	0.0	-34.7	15.9	26.4	-3.2	17.2	-2.6	27.6	17.7	-32.3	-32.3	17.6	27.4	-2.9	16.7	-3.7
PH1 3	9	-34.7	-44.1	-33.6	-3.5	47.2	-2.6	-35.4	-42.3	-32,3	-2.3	47.6	-2.6	-35.9	-43.3	-33.7
PSI	DEG	-1.3	2 • 1	-0.8	1.2	-5.8	-2.0	-1.6	-2.1	3.8	1.0	-1.1	1.5	-2.0	-0.4	0 • 4
GAMMA	r E	o 10.	φ.α	1.2	10.0	10,4	α.	0° M	7.5	3.6	4.4	13,3	P. 7	23.5	4.4	4.7
THETA	DEC	3.5	-1.2	4.5	-2.A	0.3	-n.3	1.1	G. 6.	9.0-	4.0	Ú.7	2.0	-9.3	-0.4	1.6
														7.1		
			ı		1		Į.		ı				1	8	1	
	L.		l				i						1	27.7	1	
PHIO	DEG	325.3	195.9	206.4	356.8	287.2	357.4	207.6	197.7	327.7	237.7	287.6	117.4	87.1	196.7	86.3
			ı				ļ		1		1		1			316.9
1 >	348	0.177	0.356	0.162	9.185	0.024	0.393	9.044	0.252	0.254	0.040	0.444	0.037	0.207	0.184	0.490
>	BAN	0.152	1.293	0.395	n.418	0.34R	0.222	0.052	0.411	0.038	0.282	0.434	80.0	0.75R	583	0.357
															300	400

APPENDIX B RANDOM NUMBER GENERATOR CONFIDENCE TEST

FOR RNI 11	OBS.FREQ	EXP.FRED	0-F	(n=E)++2	10-F1**2/E
PARTITION (1)	44	40	4	16.0	0.40
PARTITION (2)	38	4 n	-2	4.0	0.10
PARTITION (3)	39	4 n	-1	1.0	0.02
PARTITION (4)	40	4.0	0	0.	0 •
PARTITION (5)	45	40	5	25.0	0.62
PARTITION 1 61	45	4 n	5	25.0	0.62
PARTITION (7)	43	4 n	3	9.0	0.22
PARTITION (8)	40	4 11	0	ο.	0,
PARTITION (9)	38	40	-2	4.0	0.10
PARTITION (10)	28	40	-1.2	144.0	3,60
CHI SQUARED	5.70 V	ARIANCE	0.n787	3 MEAN	0.484644
FOR RN(2)	GBS.FREQ	EXP.FPF0	0-F	(n=E)**2	[U=F]**2/E
PARTITION [1]	-34	40	-6	36.0	0.90
PARTITION 1 21	45	4.0	5	25.0	0.62
PARTITION (3)	46	4.0	+	36.0	0.90
PARTITION (4)	36	4 n	_4	16.0	0.40
PARTITION 1 51	33	4 ņ	-7	49.0	1.22
PARTITION [6]	39	4 n	-1	1.0	0.02
PARTITION [7]	49	41	9	81.0	2,02
PARTITION [8]	37	4.0	F	9.0	0.22
	37	40	-3	9.0	0.22
PARTITION (9)			4	16.0	0.40

FOR RN(3)	OBS.FREQ	EXP.FREQ	0-F	(n=E]**2	[U=F]**2/E
PARTITION 1 11	32	4 ()	- A	64.0	1.60
PARTITION 1 21	4 0	4 n	n	n.	0 •
PARTITION 1 31	35	46	- 5	25.n	0.62
PARTITION 1 41	46	4 0	*	36.n	0.90
PARTITION (5)	36	40	- 4	16.0	0.40
PARTITION (6)	49	4 n	9	81.n	2.02
PARTITION 1 71	41	40	1	1.0	0.02
PARTITION (81	38	4 n	-2	4.0	0.10
PARTITION 1 91	38	41	- ?	4.0	0.10
PARTITION [10]	45	4 (1	5	, 25.n	0.62
FOR RNI 41	BS.FREQ	EXP.FPF0	n==	10-E1++2	[U+F]**2/E
PARTITION (1)	34	4 n	-6	36.n	0.90
PARTITION 1 21	35	4 n	- 5	25.n	0.62
PARTITION (3)	44	4 n	Δ	16.n	0.40
PARTITION (4)	33	40	-7	49.0	1.22
PARTITION (5)	38	4 (*	-2	4.0	0.10
PARTITION 1.61	49	4 0	Q	81.n	2.12
PARTITION (7)	39	40	-1	1.0	0.12
PARTITION [8]	42	4 0	2	4.0	0.10
PARTITION 1 91	49	41	Ģ	81.0	2.02
PARTITION (0)		40	_ ~	9.0	0.22

FOR PN(5)	HBS.FRE0	EXP.FRF0	7-F	(n=E)**2	[U=F]**2/E
PARTITION (1)	39	40	-1	1.0	0.02
PARTITION (2)	56	4 0	16	256.n	6,40
PARTITION 1 31	43	4 (1	3	9.0	0,22
PARTITION (4)	41	4 0	1	1.0	0.02
PARTITION (5)	36	40	-4	16.0	0.40
PARTITION 1 61	38	40	-2	4.0	0.10
PARTITION [7]	29	4.0	-11	121.0	3.03
PARTITION 1 HI	38	4 0	-2	4.0	0.10
PARTITION 1 91	45	40	5	25.0	0,62
PARTITION (110)	35	4 n	- 5	25.0	0.62
FOR PNI 6)	OBS.FREQ	EXP.FFF0		10=E1**5	10=F1**2/E
PARTITION (1)	38	4 (1	-2	4.0	0,10
PARTITION 1 21	44	4 n	- 4	16.0	0,40
PARTITION (3)	37	4 n	-3	9.0	0.22 1.22
PARTITION 1 41	47	4 11	7	49.0	0.22
	43	4 n	,3	9.0	1.60
PARTITION [5]	32	40	-8 5	64.0 25.0	0.62
PARTITION [61	AE	4.0		20.U	
PARTITION [6] PARTITION [7]	45	40		84 A	9.119
PARTITION [6] PARTITION [7] PARTITION [8]	31	4 n	-9	81.0	2,02
PARTITION [6] PARTITION [7]	-			81.0 9.0 0.	2,02 0,22 0.

FOR RN(7)	BS.FREQ	EXP.FRED	0-5	10-E1++2	[U-F]**2/E
PARTITION (1.)	41	4 (1	1	1.0	0.02
PARTITION (2)	45	4.0	E .	25.n	0.52
PARTITION [3]	36	4 0	-4	16.0	0.40
PARTITION (4)	35	4.0	- 5	25.0	0.52
PARTITION 1 51	44	40	4	16.0	0.40
PARTITION (6)	43	40	्र	9.0	0.22
PARTITION (7)	36	40	-4	16.0	0.40
PARTITION (H)	41	4.0	†	1.0	0.02
PARTITION (9)	39	4 0	-1	1.0	0.02
PARTITION (10)	40	4 n	n	0.	0.
FOR RN[8]	OBS, FREQ	EXP.FPED	0-F	[n=E]**2	[U-F]**2/E
PARTITION [1]	50	4 f1	1 0	100.0	2.30
PARTITION (2)	38	40	-2	4.0	0.10
PARTITION (3)	30	4 0	=1 0	100.0	2,50 0,52
PARTITION 1 41	35	4 0	-5	25.0	0.62
PARTITION [5]	45	4 n	5	25.0	0.40
PARTITION [6]	36	4 P	-4	16.0	0.52
PARTITION (7)	35	4 ()	-5	25.0	2,50
	50	. 40	1.0	100.0	0.90
PARTITION ! BI		4 0	-6	36n	
PARTITION [8] PARTITION [9] PARTITION [10]	34 47	4.0	7	49.n	1.22

FOR RN(9)	HBS.FREQ	EXP.FRF0	0-F	10-E1++2	[U=F]**2/E
PARTITION (1.)	45	4 n	5	25.0	0.62
PARTITION 1 21	34	4 0	-6	36.0	0.90
PARTITION (3)	39	4 n	-1 .	1.0	0.02
PARTITION 1 41	35	4.0	- 5	25.0	0.62
PARTITION (5)	41	4 0	1	1.0	0.02
PARTITION 1 61	33	4 0	- 7	49.0	1.22
PARTITION [7]	41	40	1	1.0	0.02
PARTITION (8)	43	4 0	.3	9.0	0.22
PARTITION (9)	50	4 (1	10	100.0	2.50
PARTITION (18)	39	4 n	-1	1.0	0,02
,					
FOR RN(10)	OBS, FREQ	EXP.FRED	0-F	In=E1**2	[U-F]**2/E
PARTITION (1)	46	4 (1	6	36.0	0.90
PARTITION 1 21	32	4 0	- P	64.n	1.60
PARTITION (3)	36	4 n	-4	16.0	0.40
PARTITION (4)	4 4	4 0	4	16.0	0.40
PARTITION (5)	39	4 0	-1	1.n	0.02
PARTITION (6)	48	4 0	A	64.n	1.60
PARTITION [7]	49	40	Ç	81.n	2.02
PARTITION [8]	35	4 n	-5	25.0	0.62
PARTITION (9)	36	4 0	-4	16.0	0.40
	35	4 n	- 5	25.0	0.62

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